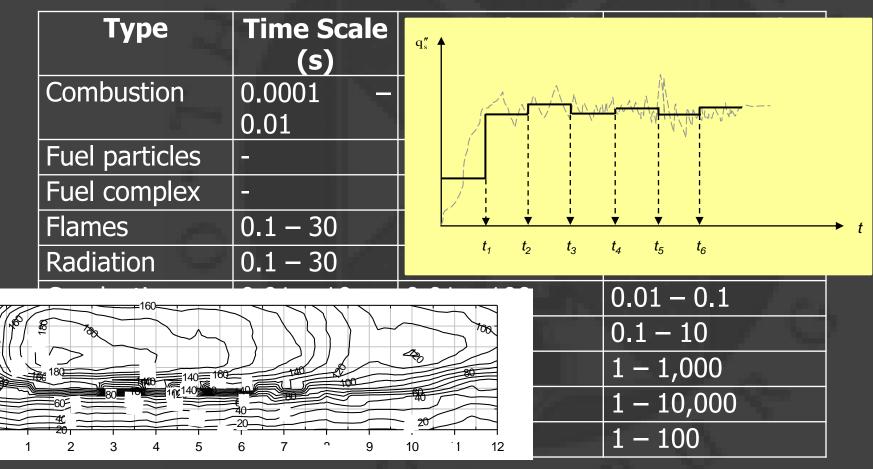
# Ignition of Solid Fuels and the Modelling of Forest Fires

José L. Torero BRE Centre for Fire Safety Engineering The University of Edinburgh, UK

# **Nodelling of Forest Fires** o**A very complex problem**

0



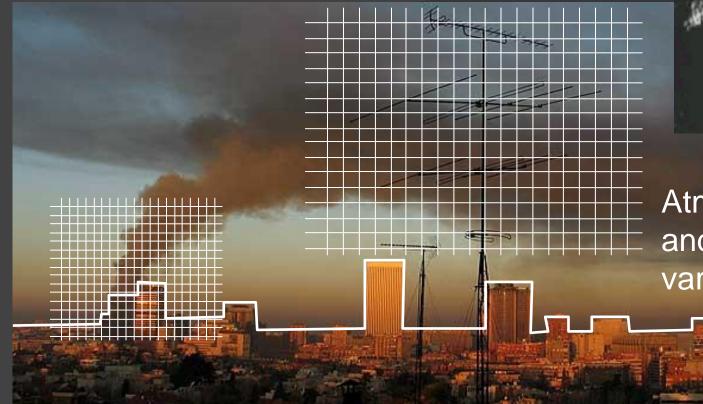
Sullivan, A., "A Review of Wildland Fire Spread Modelling, 1990-Present, 1: Physical and Quasi-Physical Models", arXiv:0706.3074v1[physics.geo-ph] (2007).

o Resolving all length and time scales in a single model is impossible

 It is necessary to develop subgrid/filtering models

 Sub-grid/filtering models need to be "compatible" with the outputs of the primary model

#### **Atmospheric Scale**





Atmospheric and terrain variables

# O Dispersion O The fire is treated as a heat source

### The Plume

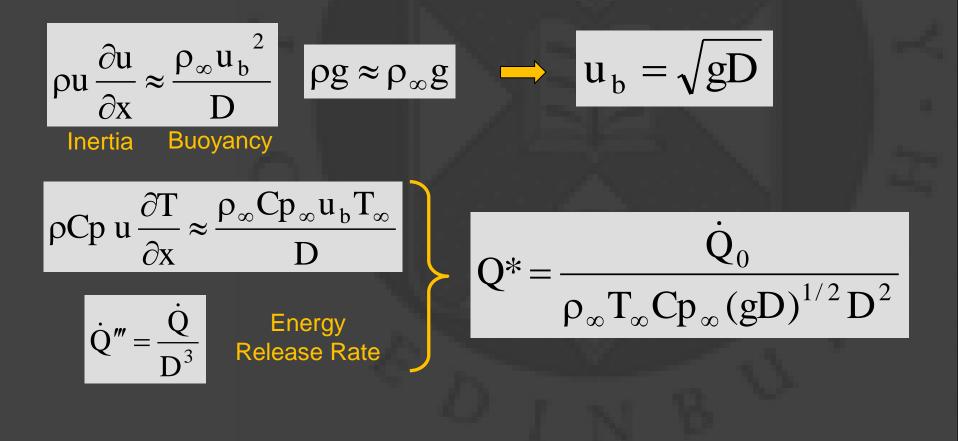
 Strength of the plume defined by the diameter of the fire (D) and the heat release rate – defines the characteristic length scale (L)

## Compatibility

#### Energy/species can be introduced within a defined volume as a source term and allowed to disperse

### **Classic Scaling Approach**

#### • Focuses on proper modelling of buoyant entrainment



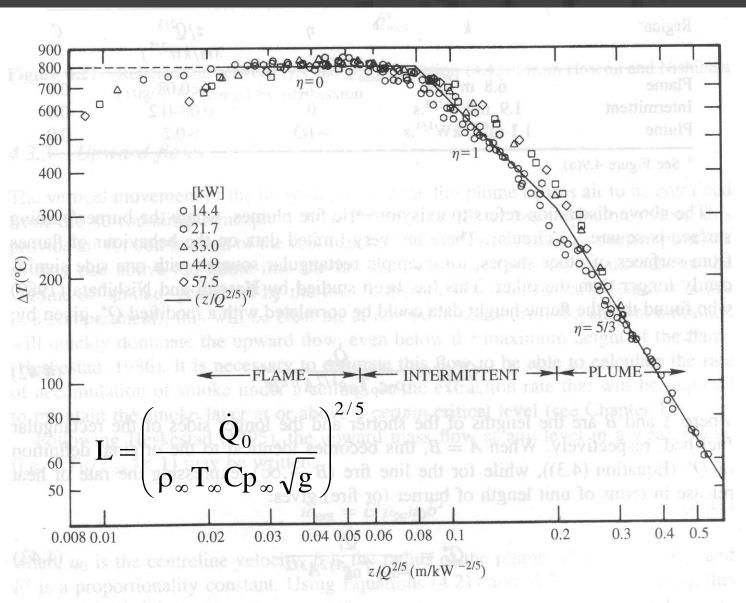
#### Use of Q\*

$$Q^* = \frac{\dot{Q}_0}{\rho_\infty T_\infty C p_\infty (gD)^{1/2} D^2}$$

#### o Obtain a characteristic length scale

• If Q\*=1  
$$L = \left(\frac{\dot{Q}_0}{\rho_\infty T_\infty C p_\infty \sqrt{g}}\right)^{2/5}$$

 Temperature and velocity fields can be presented as a function of a scaled length: Incorporates the Heat Release Rate Dependency



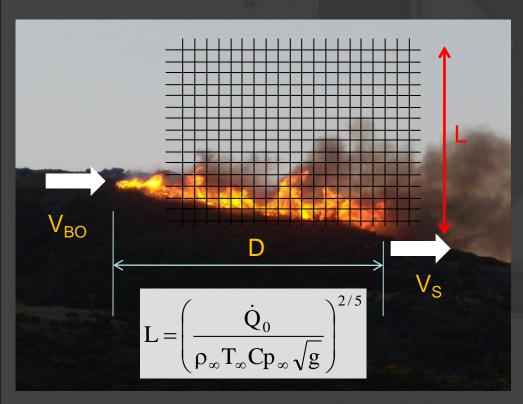
23 r

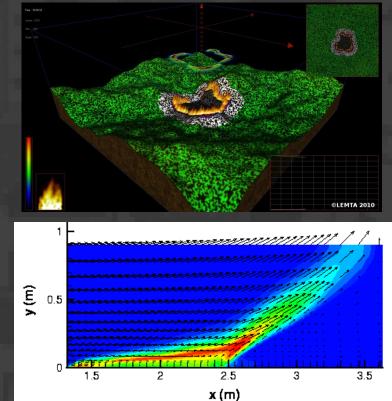
**o In a stationary fire compatibility** between the "entrainment region" and the "dispersion region" can be achieved by a sub grid model that defines the fire as a heat release rate (source term) over a well defined "volume" (characteristic length scales (D,L))

### **Flame Spread**

#### **o** Forest fires spread – thus spread rates are necessary

- Flame spread rates can be defined in an empirical way and incorporated to "atmospheric type" models – compatible
- o To maintain the characteristic length scale "burn-out" rates are necessary too compatible





### Limitations

o Flame Spread rates depend on many variables (vegetation type, density, humidity, slope, wind, etc.) o Burn-out rates depend on many variables (vegetation type, humidity, fuel load, density, wind, etc.) o Mixture of fuel and environmental variables

Fame Spread: Sequence of Ignitions 10<sup>ths</sup> of **To Atmospheric Models** Metres 10<sup>ths</sup> of

Centimetres

Ai

Gas Phase Heat Transfer

Combustion **Processes** 

**Gas Phase Heat** Transfer

Ignition

Fuel

Ignition

**Condensed Phase:** Heat Transfer + Chemistry

### Incompatible

Fuel Degradation (µm, s)

Gas Phase Chemistry (nm, ms)

Soot Production (µm, ms)

Radiative Losses (cm, ns)

Flame Temperature (cm, s)

Radiative Heat Transfer (cm, ns)

### Resolution

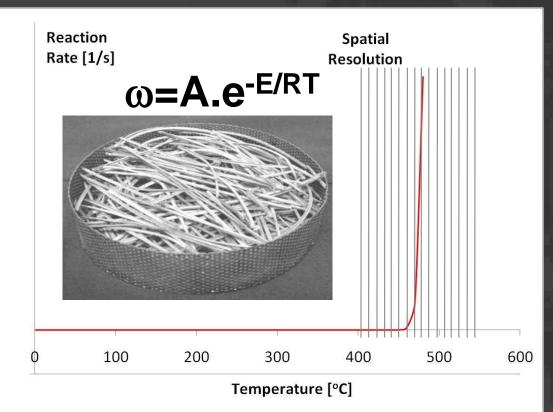
 To resolve ignition it is necessary to resolve the fasters time scales and the smallest length scales (nm, ns)

- Result needs to be fed into an combustion model (10<sup>ths</sup> cm, sec resolution)- Incompatible
- Computational cost unacceptable –
   Precision unnecessary

## Incompatibility

- Heat fluxes obtained from models/experiments (10<sup>ths</sup> cm)
- Heat fluxes applied to a porous matrix (mm)
- Temperature across the porous bed resolved (nm)
- Degradation resolved via simplified Arrhenius type chemistry (@=A.e<sup>-E/RT</sup>)
- Experimental validation studies – mass loss (cm)

#### Why nm Resolution?



 Adequate resolution of the degradation chemistry requires resolving temperature gradients within
 the fuel thickness

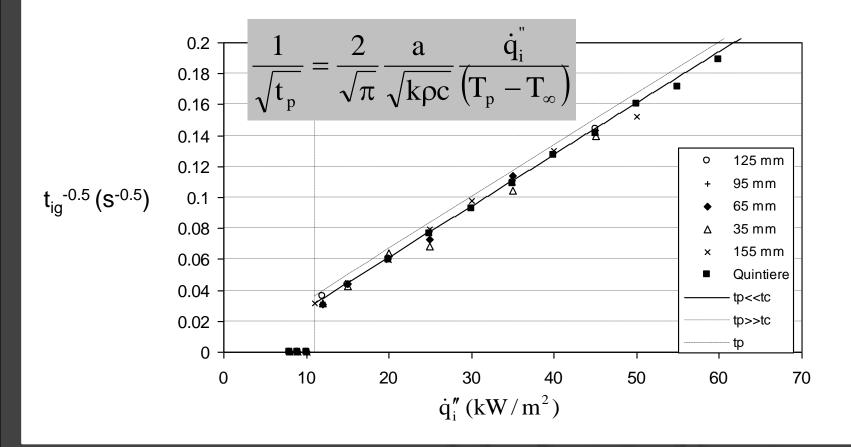
#### o Ideal Scenario:

o Input is the gas phase heat flux oSolid phase heat transfer (porous media) does not have to be resolved o Degradation chemistry does not have to be resolved oSolution: Sub-Grid model based on experimental data as an input to the

model (cm/s – resolution)

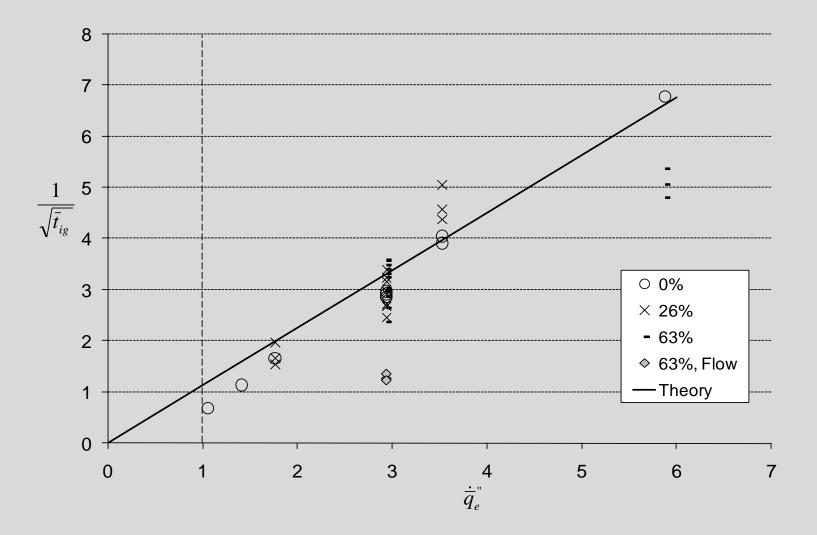
#### **Ignition Delay Time**

#### Ignition time is linearly dependent to incident heat flux (cm/s – Model)



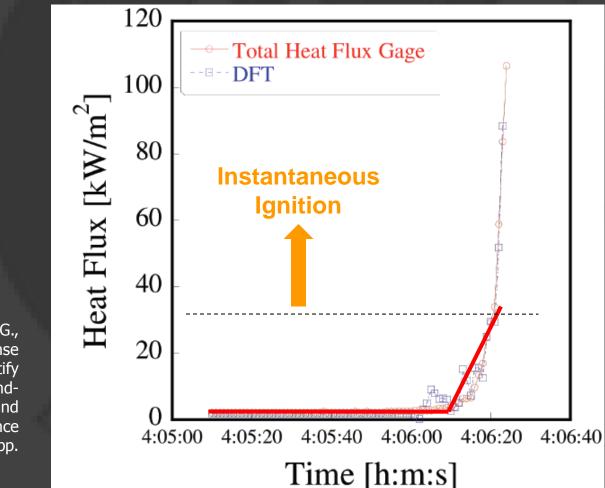
#### Quintiere, Fire and Materials, 1983

#### **Applicable to Forest Fire Fuels**



#### **Ignition Delay Times**

#### • Can be estimated as a function of the heat flux if the heat flux is a constant



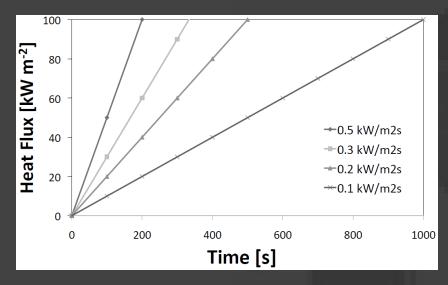
Manzello, S. L., Park, S. H., Cleary, T. G., Shields, J. R, "Developing Rapid Response Instrumentation Packages to Quantify Structure Ignition Mechanisms in Wildland-Urban Interface (WUI) Fires", Fire and Materials 2009. 11<sup>th</sup> International Conference Proceedings, San Francisco, CA, 2009, pp. 215–224.

#### Modify Mathematical Solution

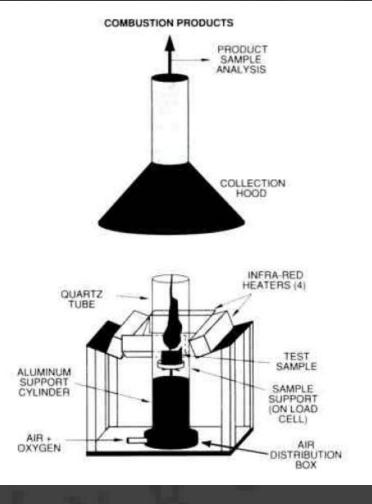
 For linear ramps integrating the expression for time to ignition over time, it can be shown that:

$$t_{ig} = \left(\frac{2}{3\,\theta_{ig}\,(\pi k\rho c_p)^{1/2}}\int_0^{t_{ig}} \dot{q}_{net}''\,dt\right)^2.$$

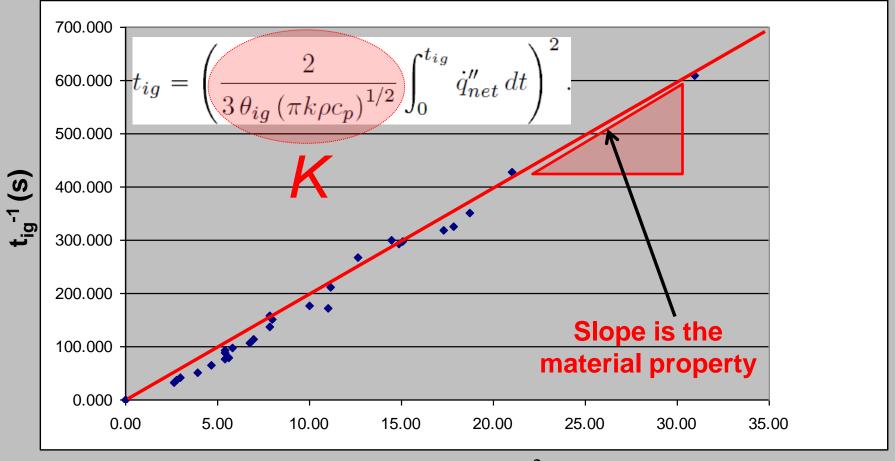
#### Validation: Fire Propagation Apparatus











$$\left(\int_0^t \dot{q}_{in}''(t) dt\right)^2$$

## Compatibility

 Combustion Model can be used to estimate the evolution of the integral heat flux to the surface as a function of time (10<sup>ths</sup> cm)

- "K"- material property (fuel type, water content, weather variables, etc.)
- o Simple model provides t<sub>ig</sub>

 No need to resolve porous media & solid heat transfer or degradation chemistry (nm)



o Forest fires cover an extensive range of time and length scales
o Different processes result in incompatible time and/or length scales

 For practical purposes, these need to be resolved with physically based sub-grid models that ensure compatibility

# Thank you